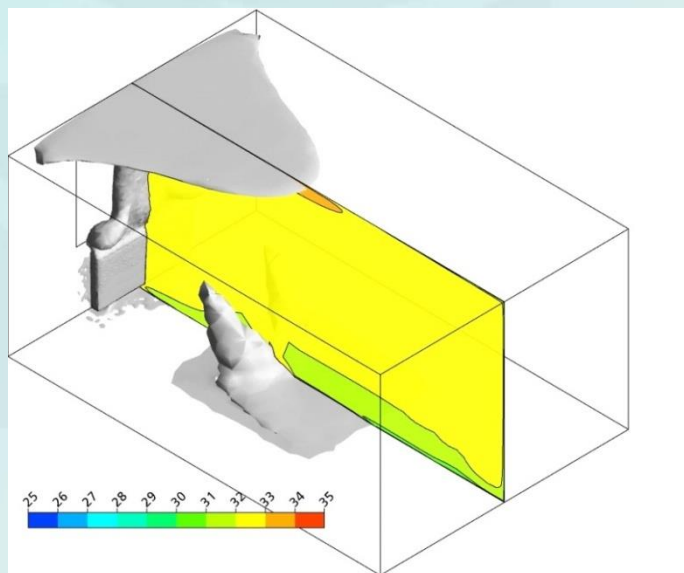


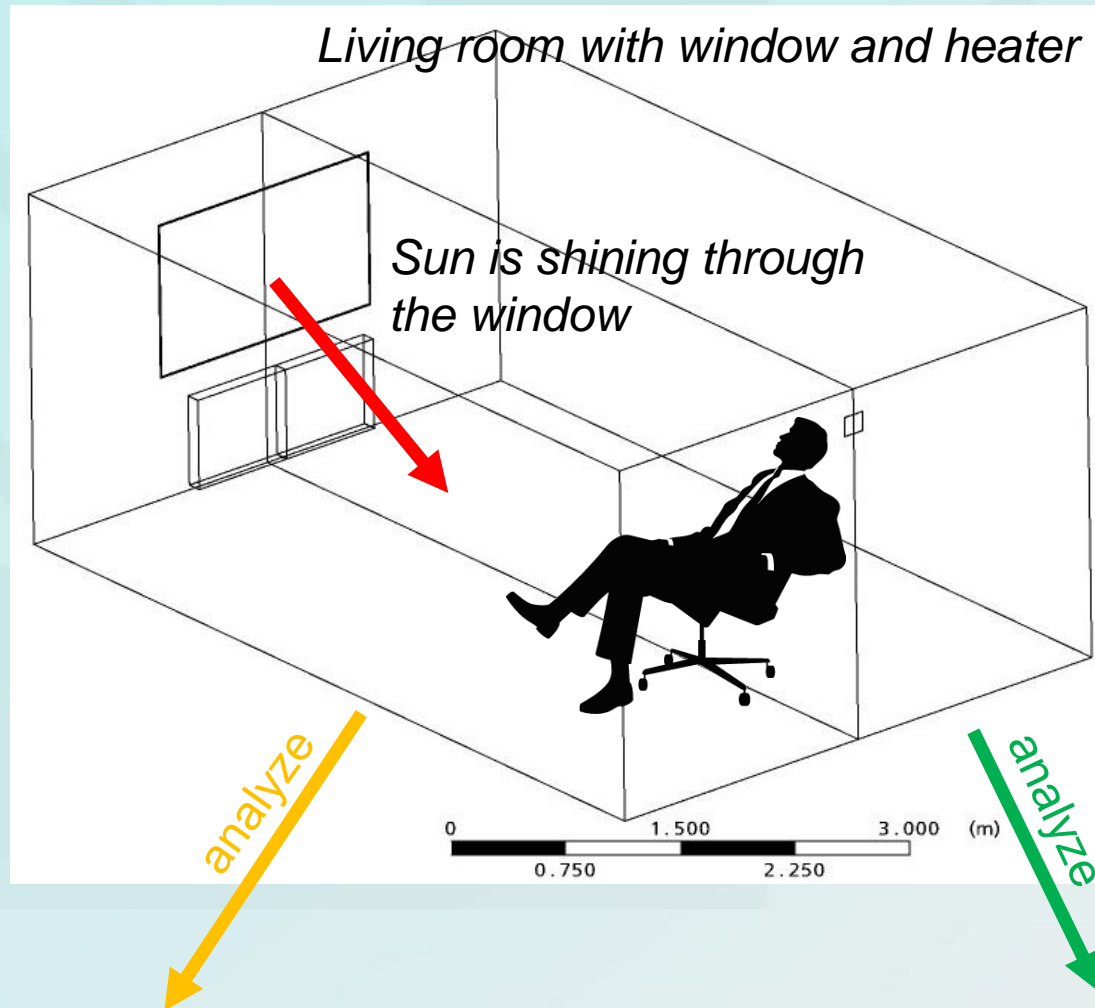
A. JAKOVIČS, S. GENDELIS

MODELLING OF HEAT BALANCE AND COMFORT CONDITIONS IN A LIVING-ROOM WITH SOLAR RADIATION SOURCE



February, 2013, Ilmenau, Germany

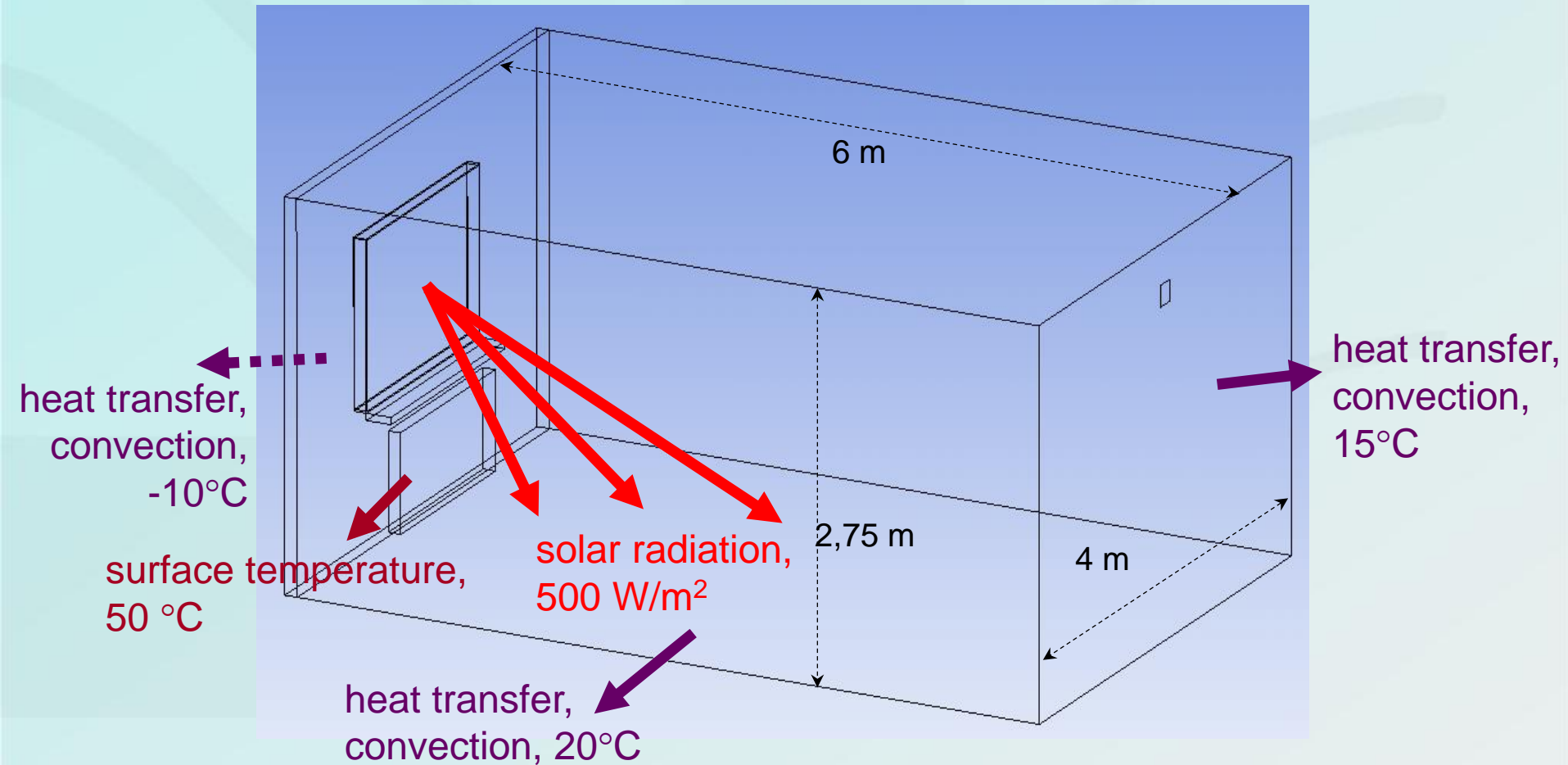
GENERAL PROBLEM FORMULATION



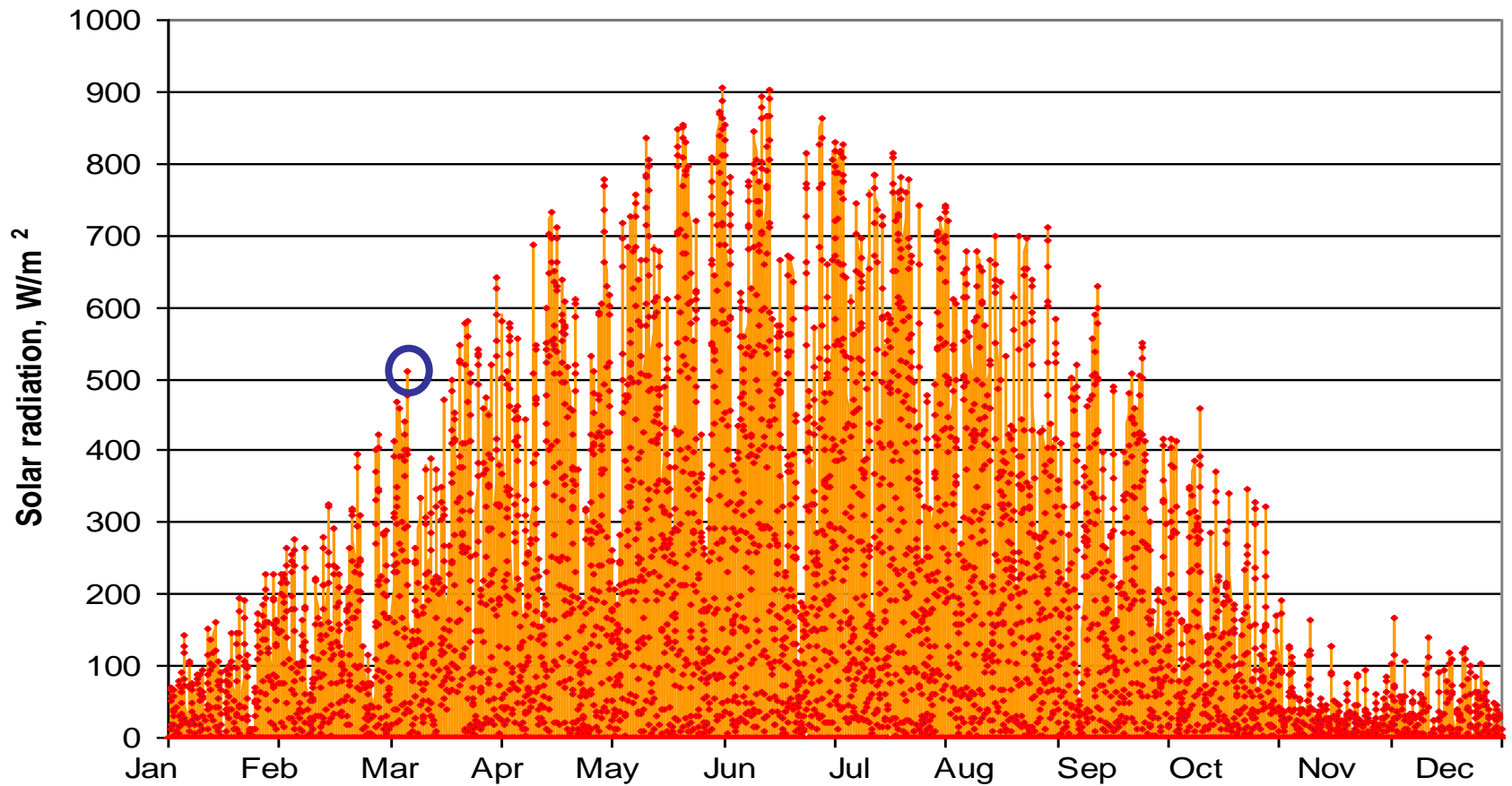
1. Heat balance of the room

2. Thermal comfort conditions

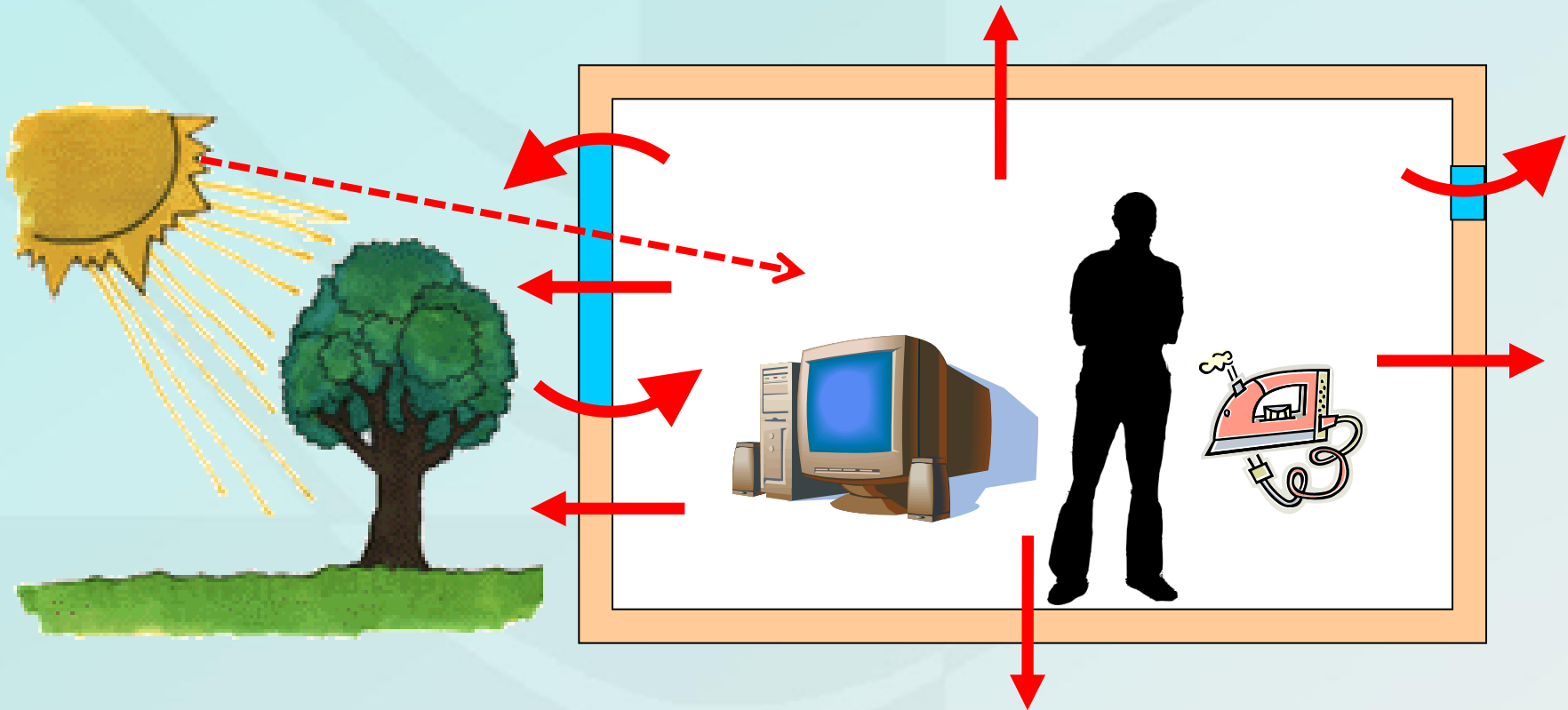
GEOMETRY, BOUNDARY CONDITIONS



SOLAR RADIATION IN LATVIA, 2009



1. HEAT BALANCE OF THE ROOM



Conduction heat losses

~~Convection heat losses~~

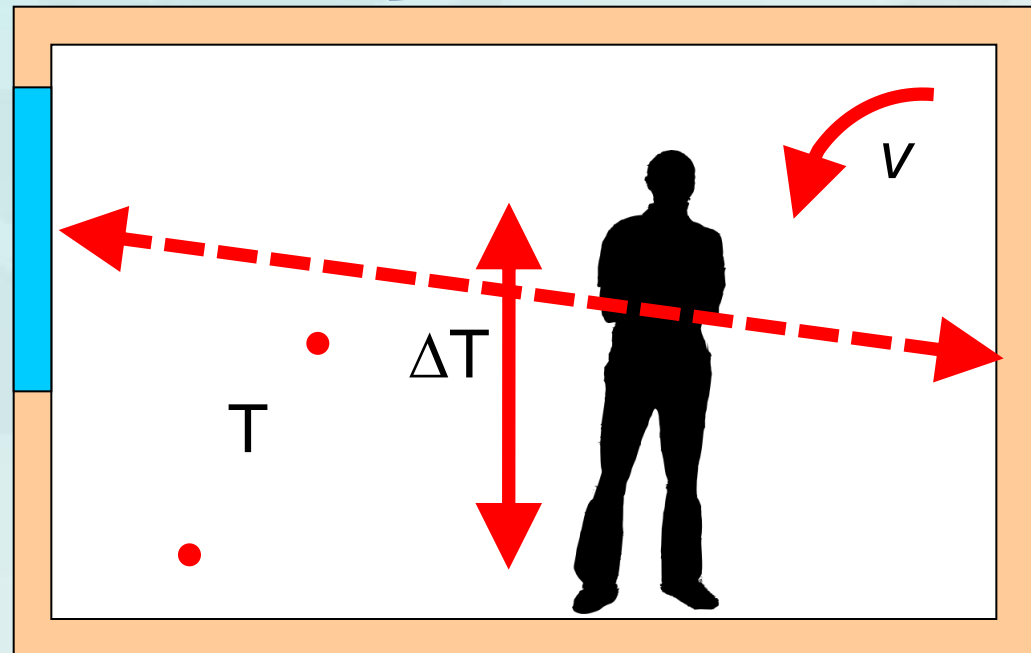
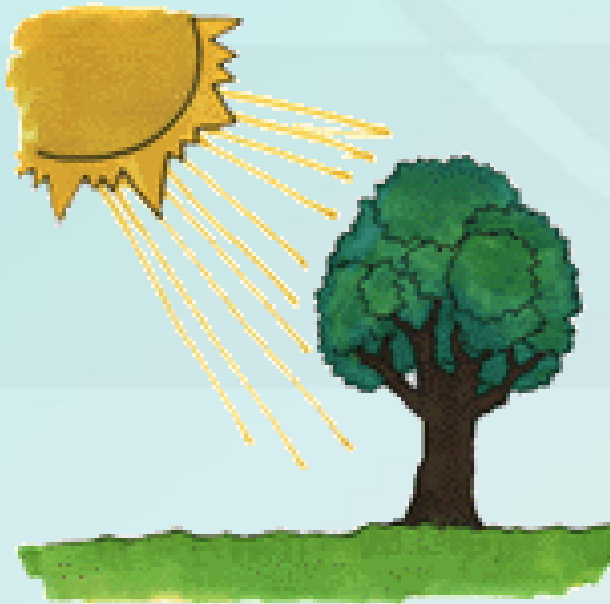
Solar heat source

~~Internal heat sources~~

2. HUMAN THERMAL COMFORT CONDITIONS

- Temperatures T
- Vertical temperature difference ΔT ,
radiant temperature asymmetry
- Air velocity v
- Other factors

Category of thermal environment **A**, **B**, **C**



CATEGORY OF THERMAL COMFORT CONDITIONS (EN ISO 7730)

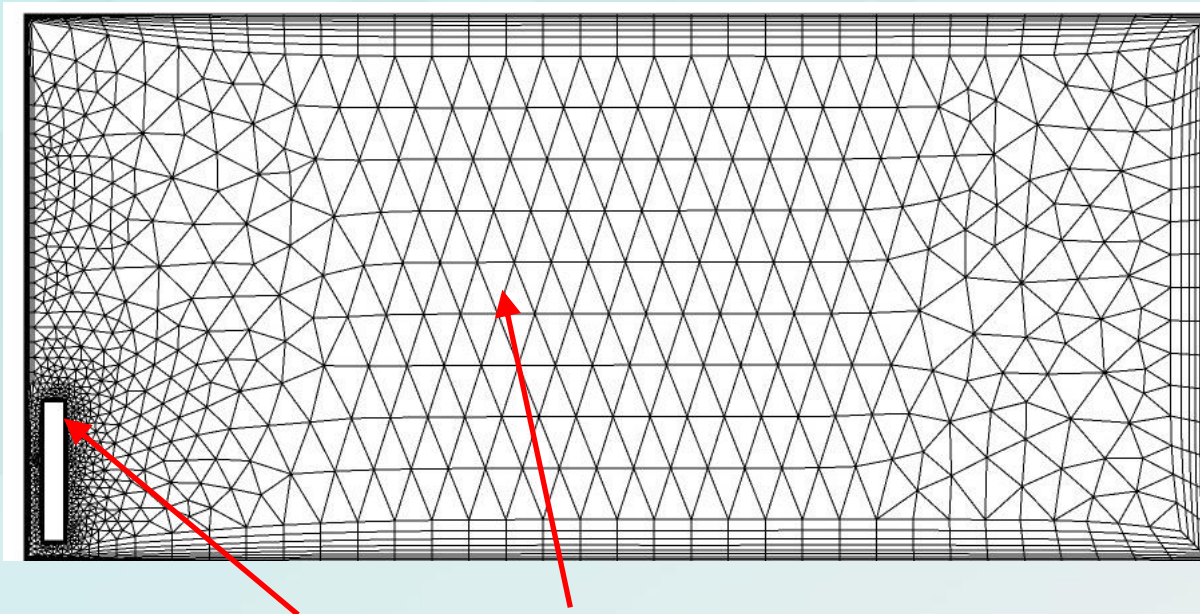
Category	Operative temperature, °C	Radiant temperature asymmetry, °C	Vertical air temperature difference, °C	Floor surface temperature, °C	Air velocity, cm/s
A	22.0 ± 1.0	<10	<2	19-29	<10*
B	22.0 ± 2.0	<10	<3	19-29	<16*
C	22.0 ± 3.0	<13	<4	17-31	<21*

* in winter season, at activity of 70 W/m². Generally depends on metabolic rate.

3D MODELLING, DISCRETISATION

For numerical calculations software *Ansys/CFX* is used with traditional differential equations:

- *Reynolds averaged momentum equation* (\vec{v} , p);
- *continuity equation*;
- *energy conservation equation* (T);
- *SST (k - ω) turbulence model*.



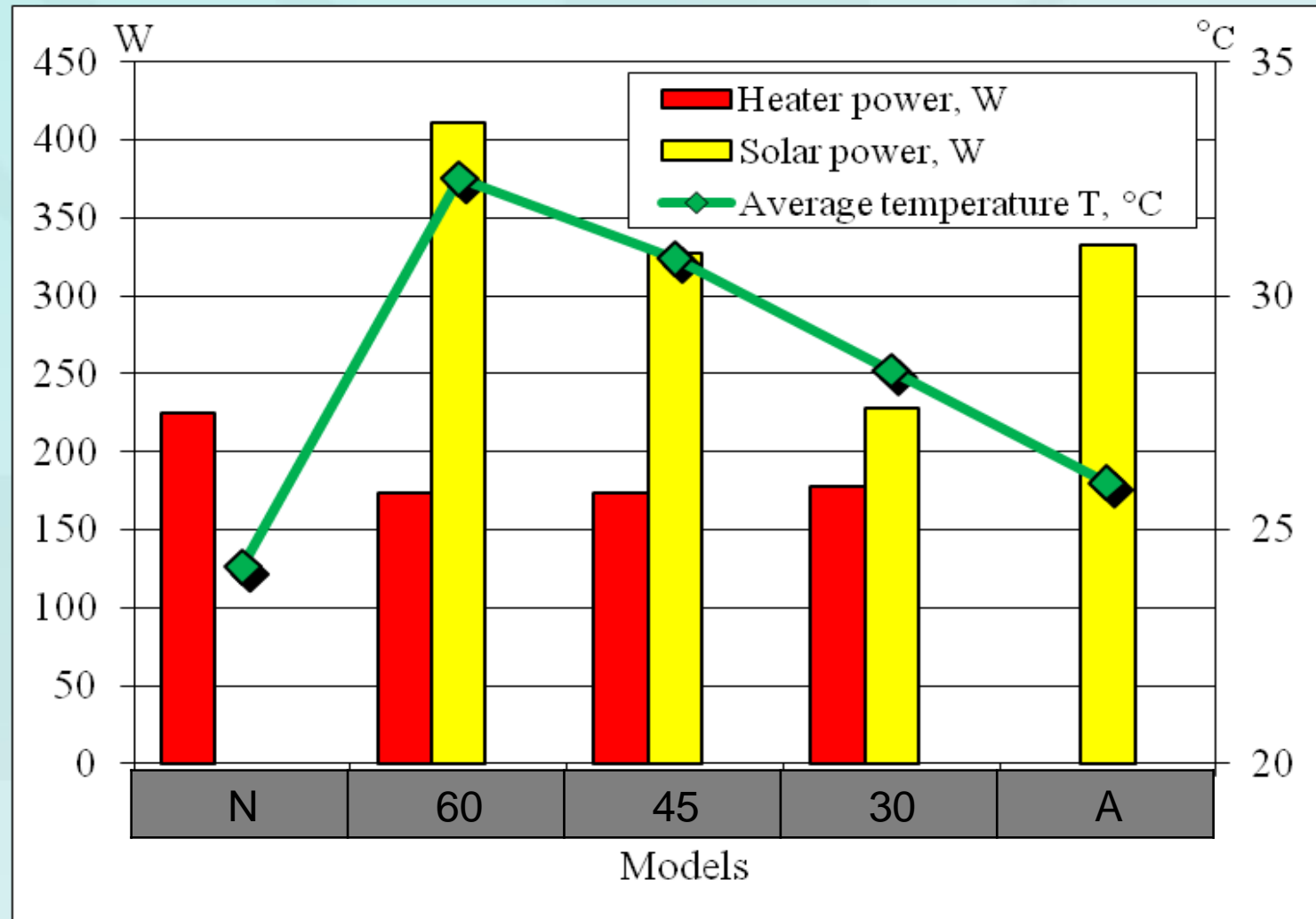
Size of elements: from **0,5 mm** till **10 cm**.

500000+ elements.

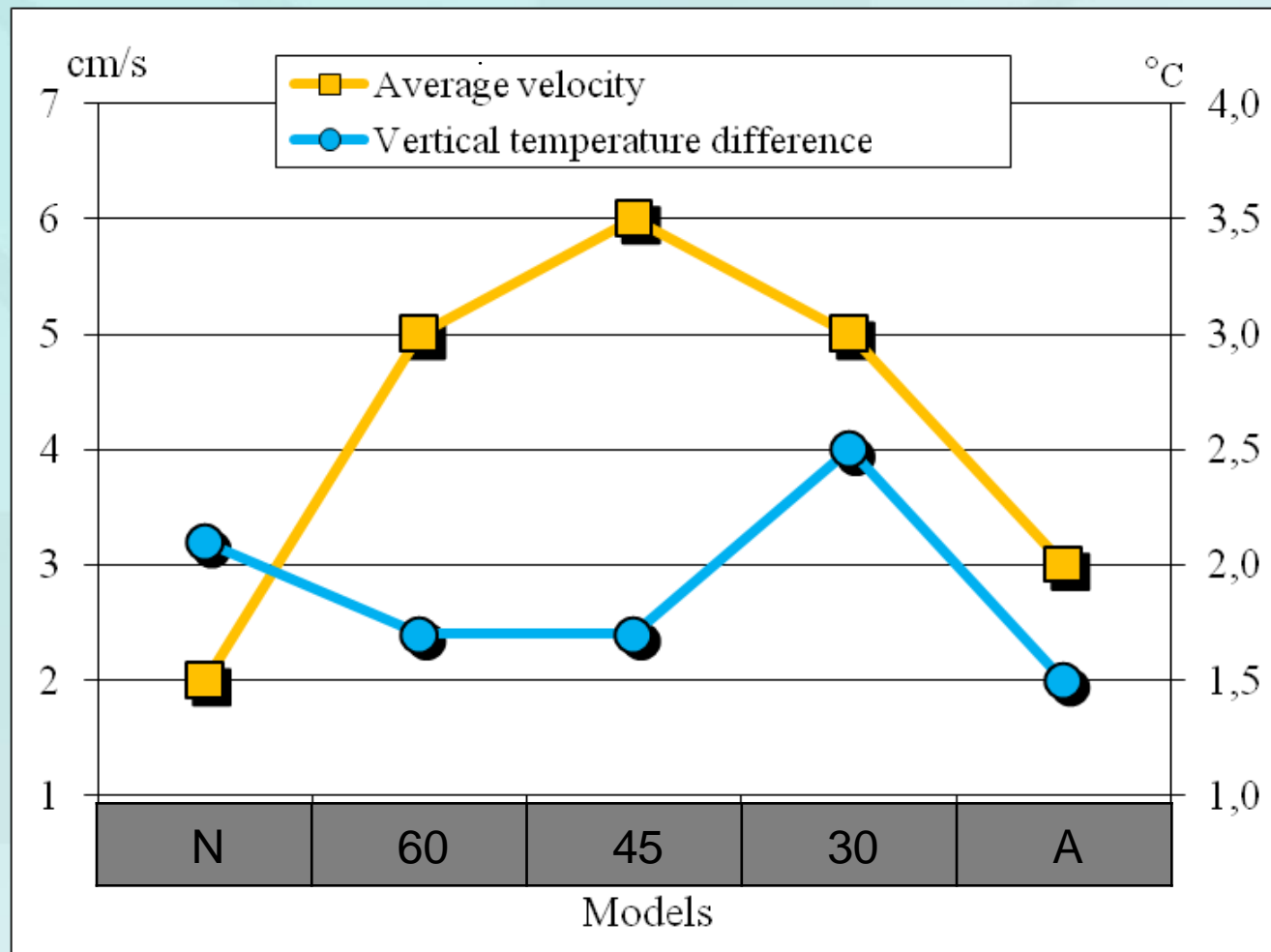
MODELLING VARIANTS

Properties	Variants				
	N	60	45	30	45
Angle of attack α (degrees)	-	60	45	30	45
Boundary condition on heater	temperature, 50°C				adiabatic
Heat amount for the heater (W)	225	173	173	178	0
Solar power W	0	411	327	228	333

1. RESULTS: HEAT BALANCE OF THE ROOM



2. RESULTS: COMFORT CONDITIONS



MODELLING VARIANTS: RESULTS

Results	N	60	45	30	A
Maximum mean air velocity v (cm/s)	4	7	8	7	5
Average temperature T (°C)	24.2	32.5	30.8	28.4	26.0
Radiant temperature asymmetry (°C)	9	12	10	10	8
Vertical temperature difference ΔT (°C)	2.0	1.6	1.6	2.4	1.4
Floor surface temperature (°C)	22	31	29	27	25
Category of thermal environment <small>[EN ISO 7730]</small>	A	C	A	B	A

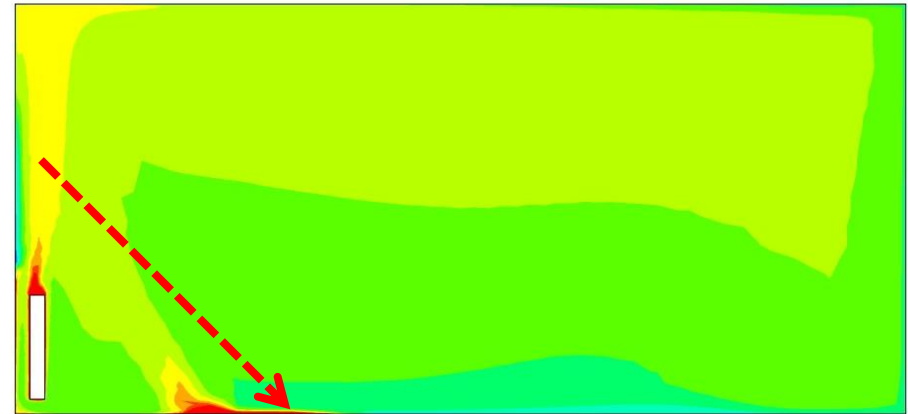
VISUALIZATION. TEMPERATURE

without solar source

with 45° solar source



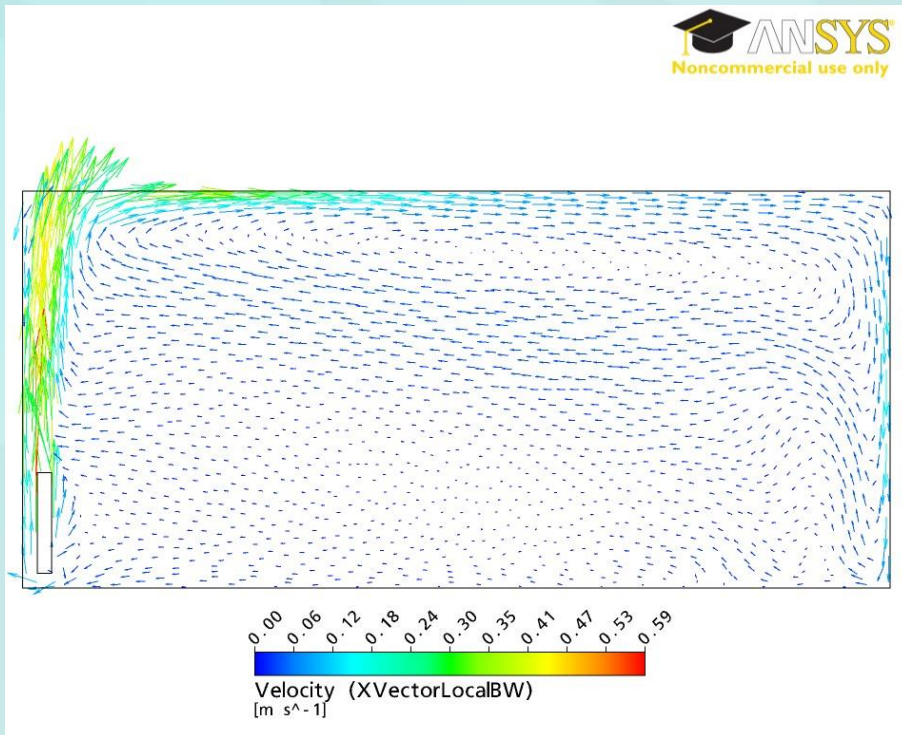
15 16 17 18 19 20 21 22 23 24 25
Temperature (XContours1525)
[C]



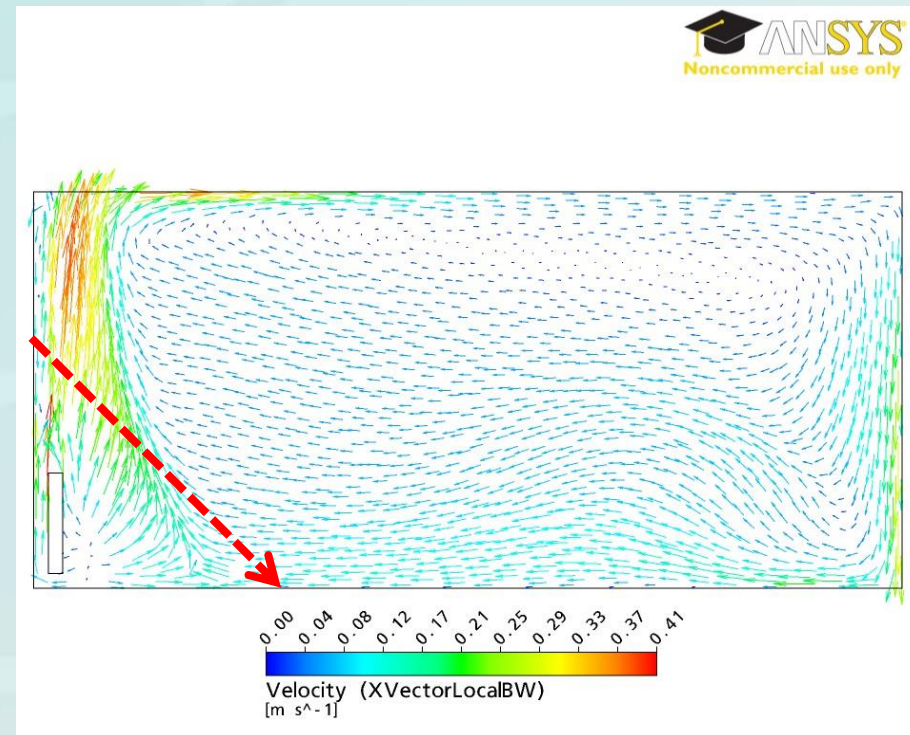
25 26 27 28 29 30 31 32 33 34 35
Temperature (XContours1525)
[C]

VISUALIZATION. AIRFLOWS

without solar source

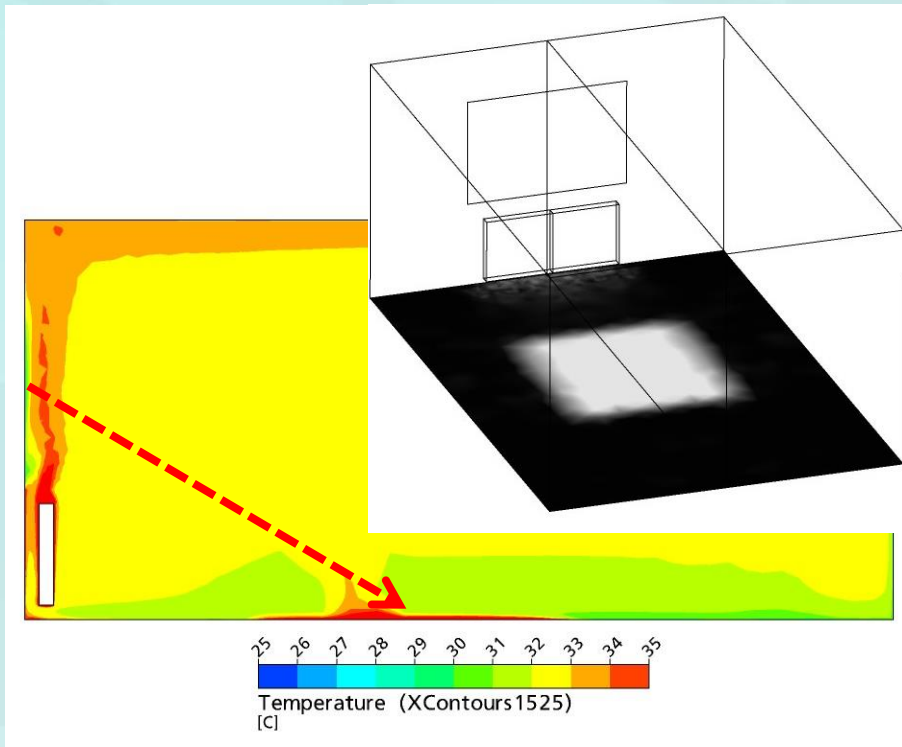


with 45° solar source

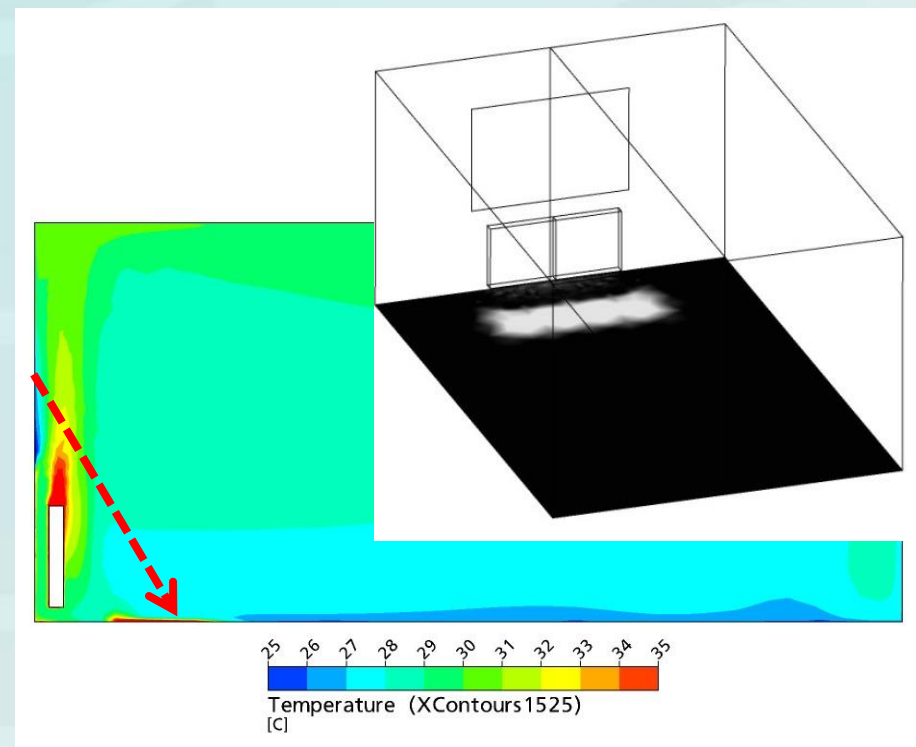


VISUALIZATION. TEMPERATURE

with 60° solar source

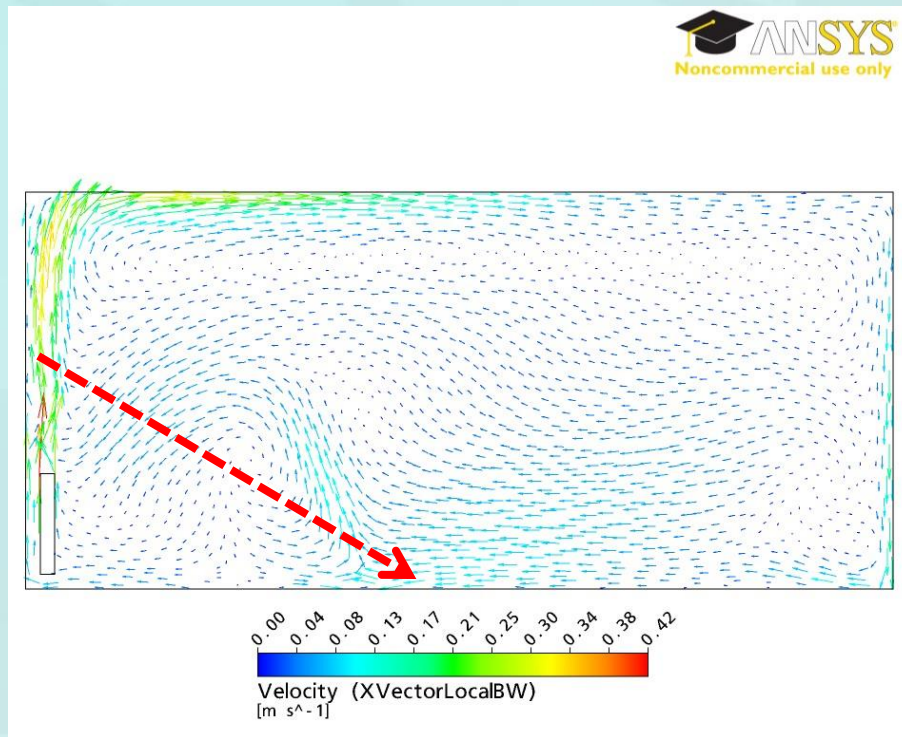


with 30° solar source

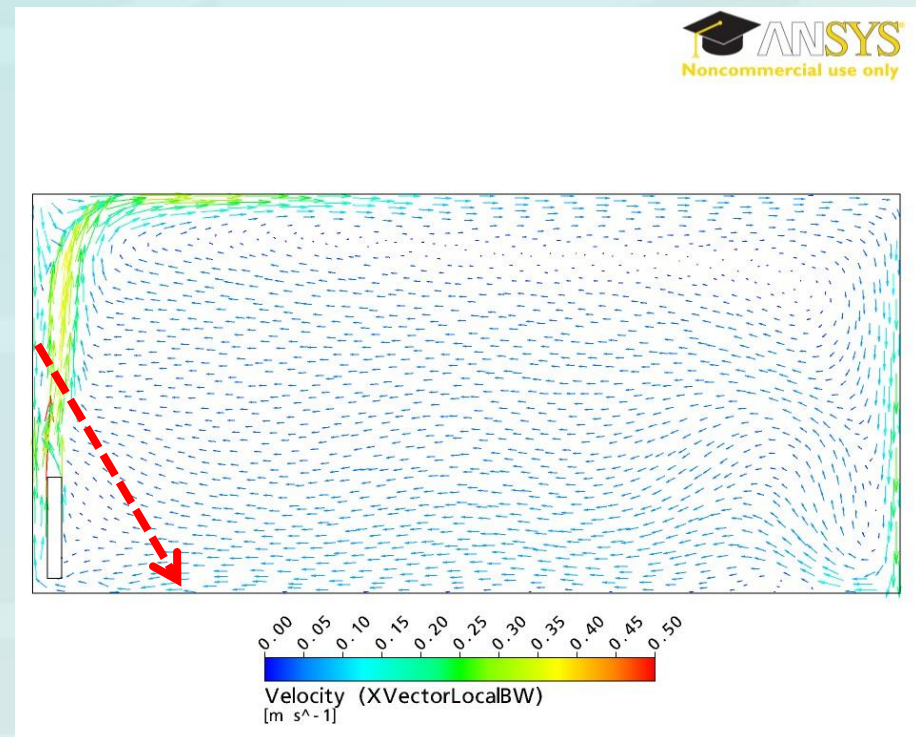


VISUALIZATION. AIRFLOWS

with 60° solar source

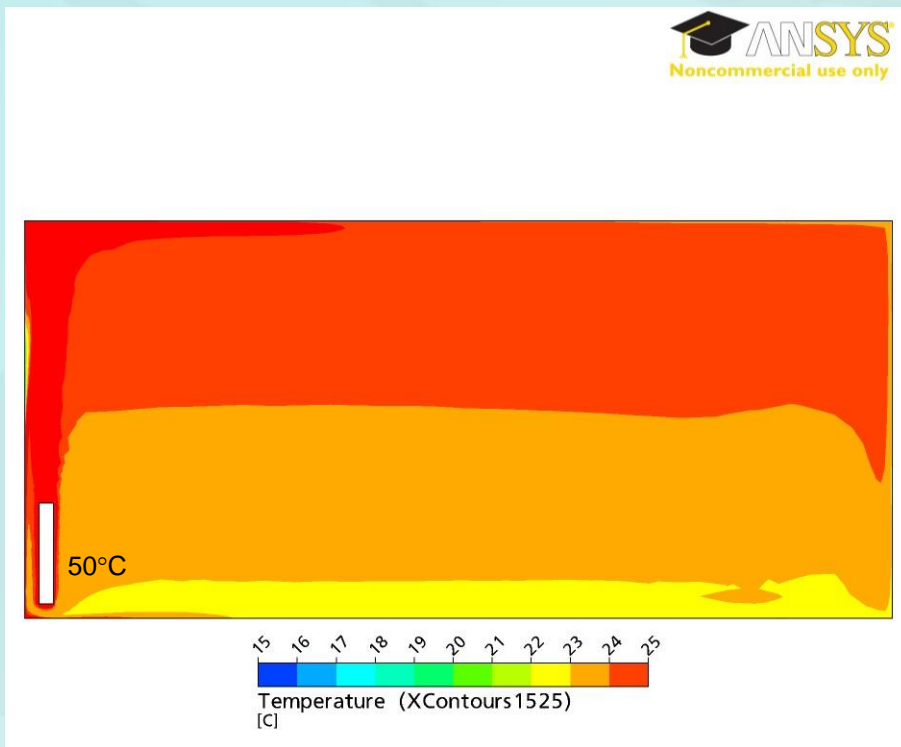


with 30° solar source



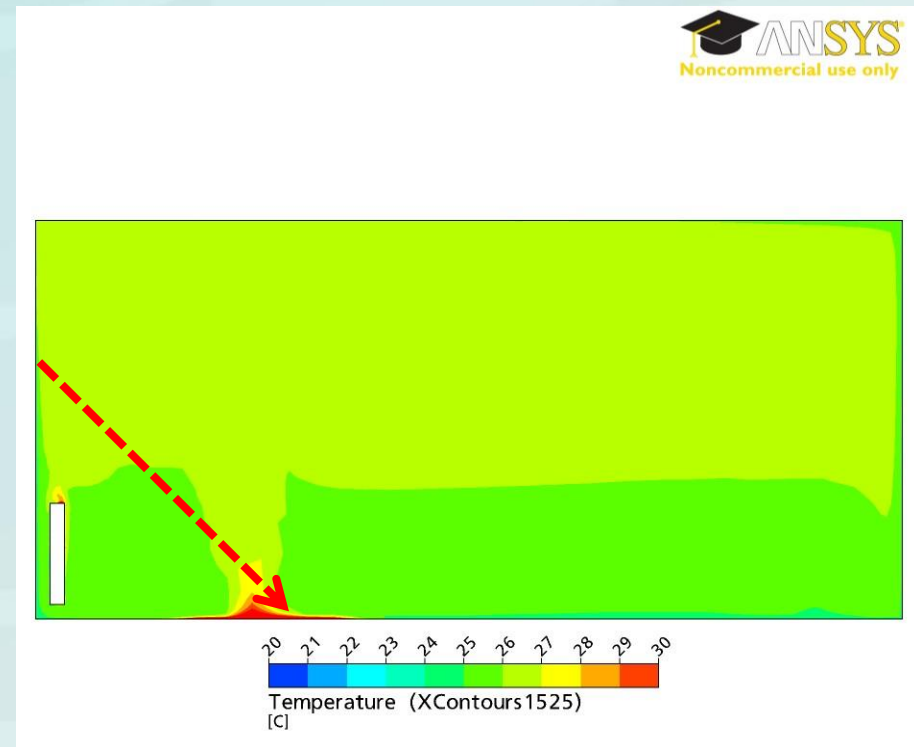
VISUALIZATION. TEMPERATURE

without solar source



$$T_{ave}=24^{\circ}\text{C}$$

without heater

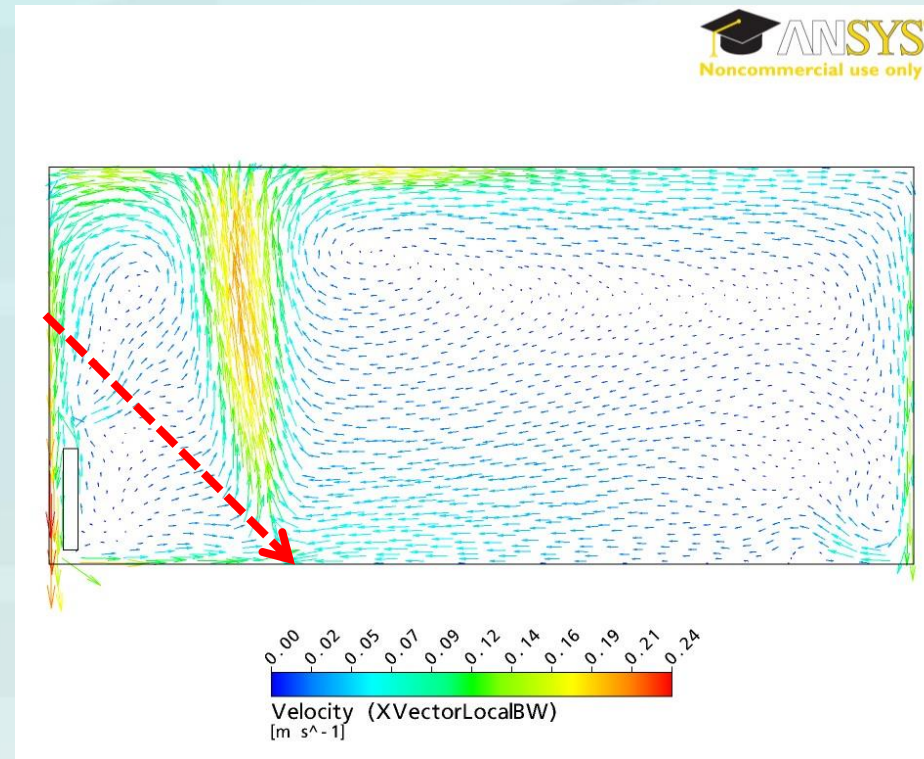
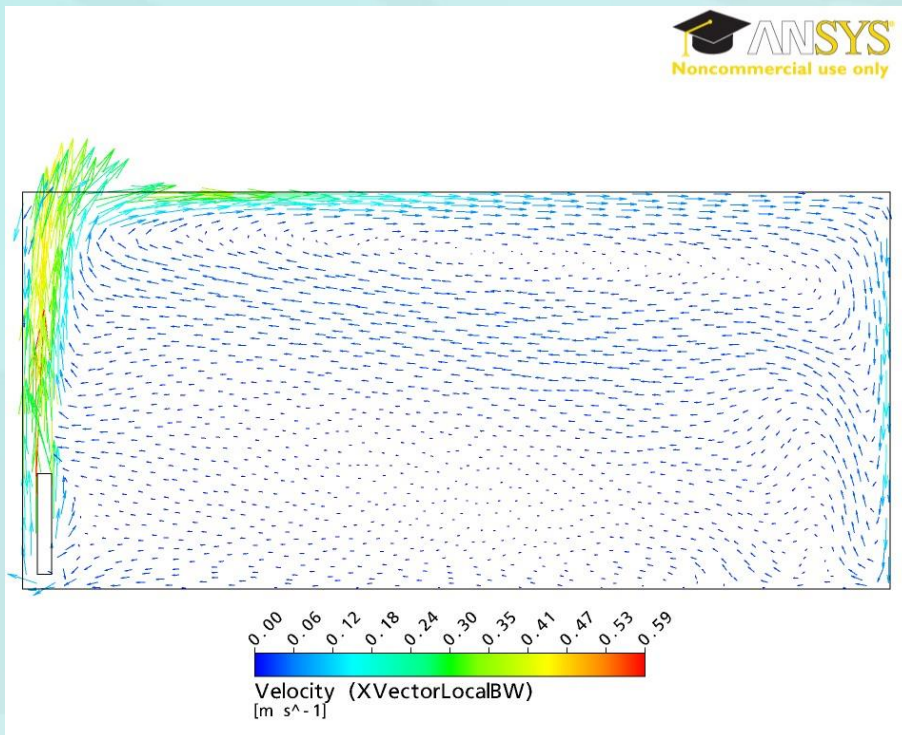


$$T_{ave}=26^{\circ}\text{C}$$

VISUALIZATION. AIRFLOWS

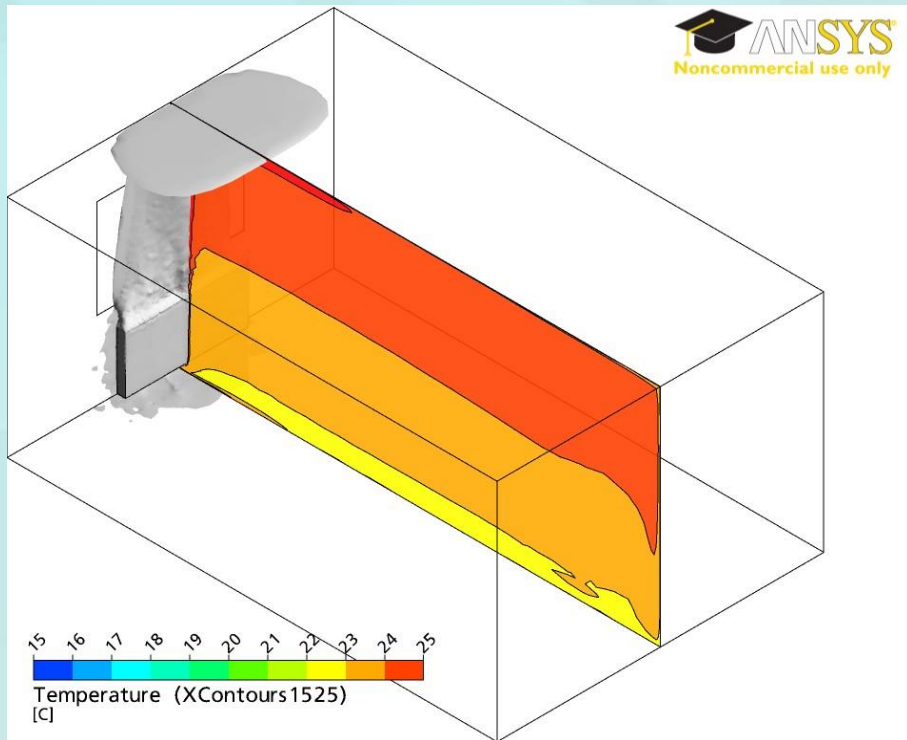
without solar source

without heater

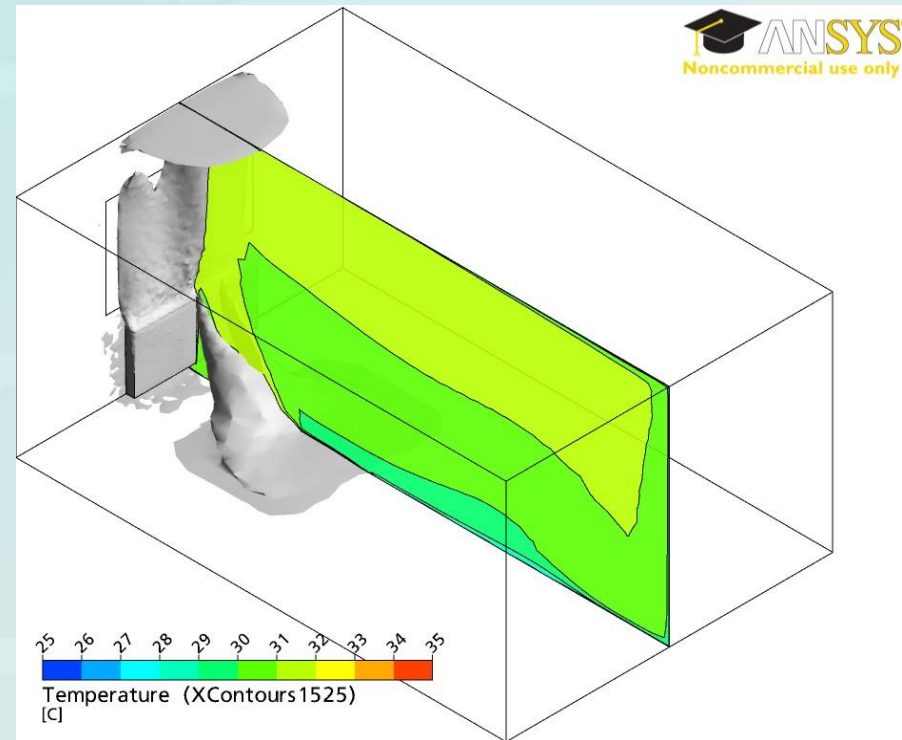


EXAMPLES OF TEMPERATURE ISOSURFACES

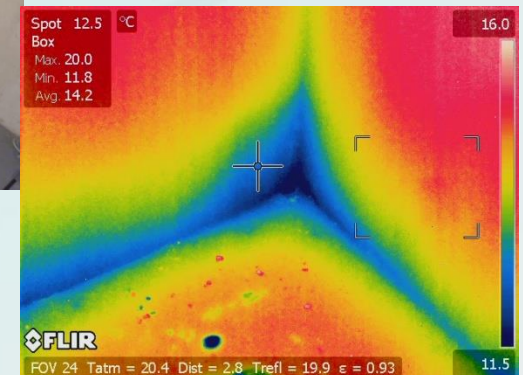
without solar source, 25 °C



with 45° solar source, 32 °C



EXPERIMENTAL VALIDATION



CONCLUSIONS

- ✓ The numerical modeling allows estimation of:
 - ✓ heating consumption of the room,
 - ✓ the temperature field, airflow distribution and the tendencies of its changes.
- ✓ Solar radiation source is very important factor. It has to be included in the numerical simulations to predict the heat balance and comfort conditions more accurately.
- ✓ The use of numerical calculations for the room at design stage allows to
 - ✓ optimize the heat balance and reduce heat losses,
 - ✓ predict the category of thermal environment for different building types.

Thank you very much for attention!

Acknowledgement: This work was supported by the European Regional Development Fund project Nr. 2011/0003/2DP/2.1.1.1.0/10/APIA/VIAA/041



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